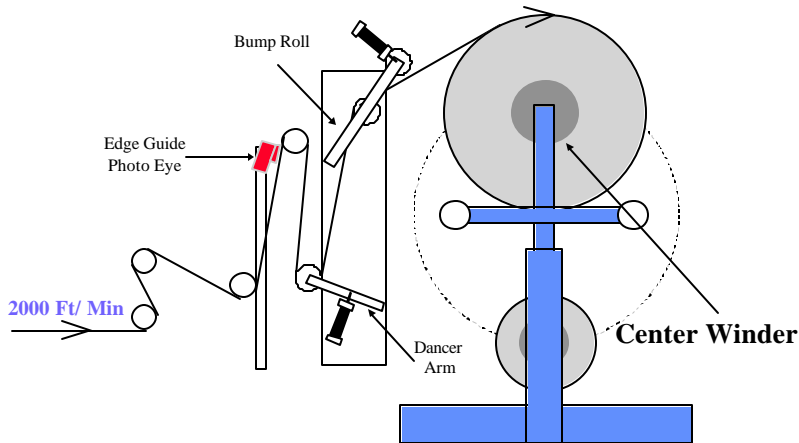


Center Winder Specification



Overview

Winding is simply a rotational means to take up and package material for more efficient handling or preparation for the next operation. A winder is used for rolling up material in a continuous or limited length of processed material such as wire, paper, film, metals and textiles.

Winders have different names in each industry.

Industry	Winder Name	Roll Name
Paper, Textile, Film	Winder	Roll
Textile	Beamer	Beam
Wire	Takeup, Reeler	Reel
Wire	Spooler	Spool
Metal	Coiler	Coil

Material Wound

Wound Materials may be extensible (Stretchable) or non-extensible (non -Stretchable). This paper will discuss both materials. The typical materials that would be wound are:

- Paper
- Metals and Foils
- Textile
- Fiber Glass Materials
- Plastic Film
- Wire and cable

Winder control

The motor RPM of the center wind varies with roll diameter and line-speed. The motor torque (which produces material tension) also varies with roll diameter and tension set point. In any given system, it is possible to **control only one variable at a time**. Center wind drives are broken up into two basic groups based on function. Either the center wind controls the speed of the material (*Speed regulated*); or the center wind controls the tension in the material (*Torque regulated*).

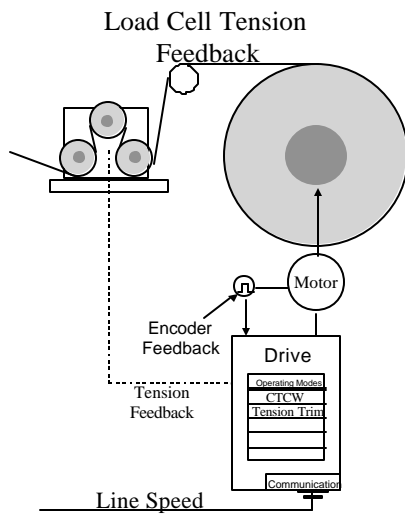


Figure 1

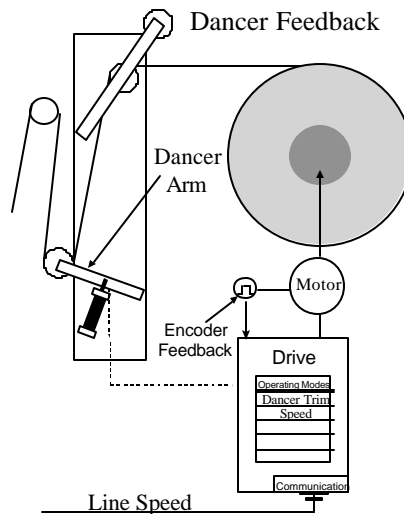


Figure 2

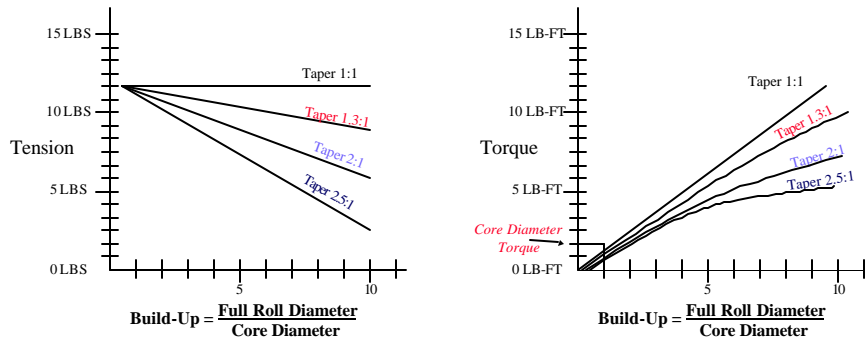
The *Torque regulated* center wind has the ability to run “open loop” or “closed loop”. Closed loop systems usually have an outer loop regulator with a feedback device typically comprised of a load cell (force transducer). *Reference figure 1 above*

The *Speed regulated* Dancer trim center winder runs in closed loop with the outer loop regulator being a dancer feedback device. *Reference figure 2 above*

Tension Control

In order to have repeatable perfect rolls, the winder must wind material under conditions of constant tension, or tapered tension as the roll builds from core. In materials such as paper, film, and textiles, the material must have *taper** so wound rolls do not become distorted. Without a *taper* tension profile, the tension on the inner layers becomes excessive and may deform the roll. Too much taper may entrap air and cause a roll deformation known as “Starring”, due to slippage between layers.

Tension trim provides direct tension control for a center wind drive by providing a trim adjustment to a torque control or a speed trim to a dancer control. Tension is measured and controlled by means of force transducers, (load cells) or dancer feedback directly actuated by the web tension. The controller will automatically adjust the motor speed to compensate for roll build-up, and compensate for changing roll inertia. Under normal operating conditions web tension is controlled, however speed follower control without trim is also provided for setup operation or as a maximum speed limit in the event of a web break.

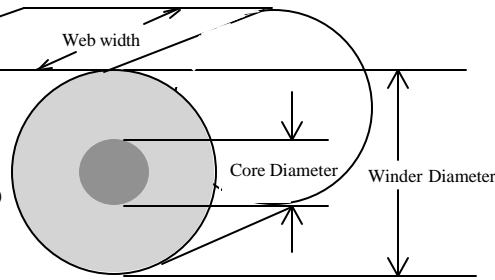


Without taper tension, rolls tend to become distorted in shape due to the increasing torque transmitted through the inner layers of the material. The graph above shows how the tension of the material should decrease at the programmed rate.

Sizing Example

Example:

Web width = 72 inches
 Core Diameter = 6 inches
 Max roll diameter = 30 inches
 Line Speed = 2000 ft/ minute
 Tension = 4 PLI (lbs per linear inch)
 Taper 2:1



$$\text{RPM} = \frac{\text{Line Speed (Feet/Min)}}{\pi * \text{Diameter}}$$

Diameter = Diameter of winder Ft

Steps

$$\text{Torque} = \text{total tension} * \text{radius}$$

Total tension = total tension at Diameter

$$\text{① Tension} = 72 \text{ inch} * 4 \text{ PLI}$$

$$\text{Tension} = 288 \text{ lbs}$$

Radius = Diameter of winder/2

Total tension = PLI * Web width

$$\text{HP} = \frac{\text{Torque} * \text{RPM}}{5250}$$

Torque = lb * Ft

$$\text{② Web HP} = \frac{288 \text{ lbs} * 2000 \text{ ft/minute}}{33,000}$$

$$\text{③ Web HP} = 17.45 \text{ HP}$$

$$\text{Web HP} = \frac{\text{Total Tension} * \text{Line speed}}{33,000}$$

Line Speed = Ft / Minute
Tension = lbs

$$\text{Winding HP} = \frac{\text{Web HP} * \text{Build up ratio}}{\text{Taper Ratio}}$$

$$\text{④ Winding HP} = \frac{17.45 \text{ HP} * 30 \text{ inch} / 6 \text{ inch}}{2}$$

$$\text{⑤ Winding HP} = 43.6 \text{ HP}$$

$$\text{Total HP} = \text{Winding HP} + \text{Losses}$$

Losses = 10 % * Winding HP

$$\text{⑥ Total HP} = 43.6 \text{ HP} + 4.3 \text{ HP} = 47.9 \text{ HP}$$

The horsepower rating of a constant torque drive for a 5:1 build up must be five times the actual required web horsepower (17.45HP) in order to produce the required torque at full roll (87.25 HP). Now we take into consideration the taper requirement, which is 2:1. The required HP drops to 42.6 HP. The energy loss in the winder is typically 10% of the winding hp.

* Reduction of web tension as winder roll diameter increases.



Customer Data

Company Name	<input type="checkbox"/> End user	<input type="checkbox"/> Distributor	<input type="checkbox"/> OEM
Contract Name #1	Contract Name #1 e-mail		
Contract Name #2	Contract Name #2 e-mail		
Address	City		
State	Zip		
Phone	Fax		

Machine Data

Type of material (paper, Te xtile, Plastic Film, Metals and Foils) _____

Machine Design speed _____ (Feet/ Minute¹) Machine Design Core Diameters _____ inches

Machine Design Max roll Diameter _____ inches Machine Design Max roll width _____ inches

Web Width _____ Inches Acceleration time _____ Sec deceleration time _____ Sec

Machine Design Tension _____ PLI Taper Ratio _____

Roll inertia _____ LB*FT²

Drive Data

Manufacture _____ Model # _____

Horse Power _____

Winder Drive New Application Retrofit

Existing Voltage 230VAC 460VAC 575VAC

Existing Drive system AC drive DC drive

Motor Data

Existing motor Manufacture _____ Model # _____

New motor required Yes No

Existing motor full load ratings: _____ AMPS

_____ Volt

_____ RPM (850, 1150, 1750)

Conduit Box location *if motor is to be replaced* F1 F2 F3 or NA

Existing Blower Motor. _____ Voltage, _____ Amps or NA

Existing Encoder Manufacture _____ NA

Existing Encoder Digital Analog AC Analog DC
 Existing Encoder Manufacturer. _____ NA
 Resolution Existing (PPR) _____ OR Volts/RPM _____
 Encoder Pickup Optical Magnetic pickup

Existing Gear Box

Gear Box Ratio _____
 Existing Gear Box Manufacture _____ Model # _____
 New Gear Box required Yes No
 Existing GearBox ratings: _____ Gear Box Ratio
 _____ Frame Size
 _____ C Face

Drive Enclosure information

Ambient Temperature in control room _____ °F or _____ °C
 Existing Drive Enclosure NEMA 1 NEMA 12 NEMA 4X AIR CONDITIONING
 New Enclosure Spec NEMA 1 NEMA 12 NEMA 4X AIR CONDITIONING
 Enclosure options Duplex outlet Lights Empty cabinet for future use
 Other _____

Existing Power Distribution⁴

Isolation Transformer _____ KVA Primary Voltage _____ AC Secondary voltage _____ AC
 Line Reactors Impedence _____ (%) Load Reactors Impedence _____ (%)
 Dynamic Braking Resistor: Duty Cycle i.e. 3%, 5% _____ % Resistance _____ Ohms
 Dynamic Resistor Power rating _____ Watts

Drive Communication Requirements

¹ Ft/minute Max RPM * π * Core Diameter (Ft)

⁴ The existing power distribution is required if MagneTek is providing a drive system

Modbus Plus Modbus Device Net Profibus Arcnet LAN Other _____

Drive Input Requirements

Start Stop Forward Reverse Run
 Jog Taper on Preset Speed1 Preset Speed 2
 Other _____

Drive Output Requirements

Drive alarm fault Drive severe fault Run Zero speed
 At speed Encoder feedback pass through (PGX card)
 Other _____

Y

Analog Input

speed reference 0-10VDC 4-20ma Other _____

Analog Output

Drive Speed (Ft/minute) Bus Voltage Other _____

Special Types of Control

Drive system start Drive system stop Regenerative to fast stop - full current limit or ramped
 DC Bus Over Voltage Suppression (Used to prevent overvoltage tripping from an unbalanced load)
 In Window, or OK to feed product. Counter for # of parts produced
 Existing load cell information
 Existing Dancer information
 Other _____

Comment [mkm1]: May need Analog output to replace the individual pump pressure sensors.